AMENDMENTS TO THE SPECIFICATION

Please amend the follow paragraphs as indicated.

[0020] Continuing with FIG. 2, the primary support system is the bearing which comprises the bearing cone 208 and the surrounding bearing seat 210. The cone and seal define facing surfaces 212, 214 which are separated by gap 216 typically of about five microns (although this dimension is representative of example only; it may be greater or less depending on the tolerances achieved by parts and assembly methods). Fluid fills this gap 216, and is maintained under pressure within this gap during relative rotation of the facing surfaces, by grooves 218 on the surface of the bearing cone 208 [[608]].

[0025] When the shaft is at rest, the oil settles into the groove annulus 264 but most of it a does not pass further down the shaft because of the capillary effect of diverging external annulus and the small radial gap between the shaft and rotating sleeve. When the motor spins up, the relative rotation of the shaft [[500]] 200 and the surrounding section 264 of the grooved pumping seal create the pressure gradient indicated by the arrow Δp, 268. This causes oil or fluid to flow out of the grooves into the quiet zone 262 as indicated by the arrow 268. Further, because of this quiet zone 262, there is no pumping action to force the oil out of the quiet zone of the grooved pumping seal into the fluid dynamic bearing, so that the necessary fluid for effective operation of this grooved pumping seal is maintained. Any air bubbles which appear in the quiet zone would tend to be captured by the pressure gradients in the fluid dynamic bearing and be expelled from the distal end of the shaft through the centrifugal capillary seal; any further air bubbles which moved into the grooves of the grooved section 264 could be expelled into the interior section of the motor because of the strong pressure gradient created by the rotation of the grooved pumping seal.